Rigid Motion Mesh Morpher (R3M): a novel approach for mesh deformation

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June 4, 2014

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- Brief introduction to R3M



5 Conclusions / Next Steps / improvements

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Trashing of the mesh at the end of each optimization cycle and generation of a new one. Time-consuming, gradient consistency lost from one cycle to the other. In some cases manual intervention during mesh generation.

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The basic idea

The internal nodes of the mesh should gracefully follow the movement of boundary nodes, as indicated by the optimization algorithm.

Existing mesh morphing methods

Method	Shortcomings
Spring analogy	Not robust
Laplacian smoothening	More robust. No
	rotation. No mesh
	anisotropy.
Linear elasticity	More robust but mesh
	anisotropy?
Radial Basis Functions	Dense matrices, Limita-
	tions in mesh size, trade-
	off between computa-
	tional cost & implemen-
	tation simplicity

Common characteristic of the first three: they don't handle naturally mesh anisotropy.

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Stencils?

Example: A node plus its neighbouring nodes (sharing one or more cells with it).























$$\vec{e_{sj}} = \vec{v_j} - \vec{v_{sj}} = \vec{v_j} - \vec{a_s} - \vec{b_s} \wedge (\vec{x_j} - \vec{c_s})$$



$$\vec{e_{sj}} = \sqrt{w_s \mu_{sj}} (\vec{v_j} - \vec{a_s} - \vec{b_s} \wedge (\vec{x_j} - \vec{c_s}))$$

The total distortion energy of the mesh will be

$$E = \sum_{s} \sum_{j \in s} \vec{e_{sj}}^T \cdot \vec{e_{sj}} = \sum_{s} \sum_{j \in s} \|\vec{e_{sj}}\|^2$$

It serves as the distortion metric which needs to be minimized. Hence the classification of the method as "optimization based".

$$\frac{\partial E}{\partial v_j} = \frac{\partial E}{\partial a_s} = \frac{\partial E}{\partial \beta_s} = 0$$

The quadratic minimization problem of the total distortion energy of the mesh, as shown above, brings us to the following symmetric positive definite system

$$\begin{bmatrix} \mathbb{A}_{uu} & \mathbb{A}_{u(a|b)} \\ \mathbb{A}_{(a|b)u} & \mathbb{A}_{(a|b)(a|b)} \end{bmatrix} \cdot \begin{bmatrix} u \\ (a|b) \end{bmatrix} = \begin{bmatrix} P_u \\ P_{(a|b)} \end{bmatrix}$$

where the RHS consists of the boundary conditions, namely the prescribed nodes' velocities. Attempting to solve it using the Schur complement leads to two different cases of elimination:

Either
$$u = f(a|b)$$
 or
 $(a|b) = g(u)$

for instance

$$u = -\mathbb{A}_{uu}^{-1} \cdot \mathbb{A}_{u(a|b)} \cdot (a|b) + \mathbb{A}_{uu}^{-1} \cdot P_u \rightarrow (-\mathbb{A}_{(a|b)u} \cdot \mathbb{A}_{uu}^{-1} \cdot \mathbb{A}_{u(a|b)} + \mathbb{A}_{(a|b)(a|b)}) \cdot (a|b) = -\mathbb{A}_{(a|b)u} \cdot \mathbb{A}_{uu}^{-1} \cdot P_u + P_{(a|b)})$$

or

$$(a|b) = -\mathbb{A}_{(a|b)(a|b)}^{-1} \cdot \mathbb{A}_{(a|b)u} \cdot u + \mathbb{A}_{(a|b)(a|b)}^{-1} \cdot P_{(a|b)} \rightarrow (\mathbb{A}_{uu} - \mathbb{A}_{u(a|b)} \cdot \mathbb{A}_{(a|b)(a|b)}^{-1} \cdot \mathbb{A}_{(a|b)u}) \cdot u = P_u - \mathbb{A}_{u(a|b)} \cdot \mathbb{A}_{(a|b)(a|b)}^{-1} \cdot P_{(a|b)}$$

Explanation of the μ_{sj} coefficient



Figure: Squeezed/anisotropic stencil (we favour rigidity in the direction of squeeze) $\mathbb{T}\neq\mathbb{I}$

Reminder:

$$\vec{e_{sj}} = \sqrt{w_s \mu_{sj}} (\vec{v_j} - \vec{a_s} - \vec{b_s} \wedge (\vec{x_j} - \vec{c_s}))$$

Relative weight stencil \Rightarrow node (scalar coefficient):

$$\mu_{sj} = \frac{\exp\left(-\frac{\|\vec{x_j} - \vec{c_s}\|^2}{h^2}\right)}{\|\vec{x_j} - \vec{c_s}\|^2 + \varepsilon}$$

Results

R3M is/does:

- Essentially mesh-less (only needs nodes and not cell or inertial data)
- Manage intrisically mesh anisotropy and rotation.

Better weighting

Improved propagation of deformation to all layers by setting the coefficients $w_s = f(\int_t E_s dt)$ as a function of the integral of the stencil distortion energy over time.

Coupling with ESI's i-adjoint solver

for automated optimization loops

This work was funded by the EU through the FP7-PEOPLE-2012-ITN "AboutFlow" Grant agreement number:317006.